A Low Order System Frequency Response Model for Large Power System and Adaptive Load Shedding

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ABSTRACT: Now a days electrical system becomes very much large and Power demand will be increasing more and more. To satisfy the power demand nonconventional sources are incorporated in the field of electrical power generation and the system becomes Inter connected. Large power system disturbances sometimes causes interconnected system to become separated into some islands and it has either surplus of generation or load. If generation is more than it can be controlled by controlling the speed of speed governors connected with the generating station will result into change in frequency. This paper includes the derivation of simple low order system frequency response (SFR) model that can be used for estimating the frequency behaviour of the interconnected power system to sudden load disturbances. The SFR model is the simplify model to analyse system dynamics.

KEYWORDS: Frequency, rate of change of frequency, frequency response, governor drop, inertia constant, reheat time constant, regulation.

I.INTRODUCTION

The frequency deviation that accompanies system islanding disturbance is caused by the imbalance between generation and load. The basic concept of model derived here is based on the uniform or average frequency where frequency oscillations or frequency deviation can be filtered out, but the average frequency behaviour retains. There are many theories which can be used to determine the total impedance to “inertial centre” \cite{1, 2, 3}, but generally only the generator and transformer impedances are taken in to consideration. The effect of these impedance places a limitation on the amount of load that any generating unit can absorb \cite{4, 5}. SFR model neglects this limitation, assuming that the disturbance is small compared to the total rating of the island, and that the equivalent machine will able to absorb this change.

In proposed scheme, large blocks of loads are removed in the initial steps and amount of load shedding is decided by the generation deficit in the system. A typical CUFLS, discussed in \cite{6}, deals with three step load shedding scheme, where in 10\% load is shed when frequency drops to 59.2 Hz, followed by a load shedding of 15\% at a frequency set point of 58.8 Hz. An additional load shedding of 20\% is carried out, when the frequency drops to 58 Hz. As suggested in \cite{7} that the load shedding, based on frequency drop, is acceptable for the cases with generation deficiencies up to 25\% and above that the scheme will have to consider both frequency drop as well as rate of change of frequency. The adaptive under frequency load shedding schemes \cite{6, 7} have been proposed to overcome the deterministic nature of the Conventional UFLS schemes. Fundamentally, all the AUFLS schemes aim at computing the active power imbalance, following a contingency in the system. This computation has been made possible through the derivation of a simple low order system frequency response (SFR) model proposed in \cite{8}. The basic SFR model averages the machine dynamic behaviour in large system into an equivalent single machine. We can think that the separate machines connected with individual generator buses can be replaced by a single large machine that is connected with ideal shaft. As a result interconnected machine oscillations are ignored and average dynamic response is obtained.
Consider a large system in which most of the generating units are reheat steam turbine units. We require to reduce this system to one described by a minimum number of equations that will use to compute the average frequency behaviour. The dynamic performance of each rotating equipment is controlled by a separate governor by integrating the individual accelerating power.

The basic SFR model averages the machine dynamic behaviour in a large system into an equivalent single machine. This approach is to provide the minimum order model that retains the essential average frequency response shape of a system with typical time constants and active speed governing. The simplified model is based on neglecting nonlinearities and all but the largest time constants in the equations of the generating units of the power system, with assumption that the generation is dominated by the reheat steam turbine generators. This means that the inertia of the generating units and reheat time constants predominate the system average frequency response. The result is a representation of only the average system dynamics, while ignoring the inter-machine oscillations [9]. The dynamic performance of each rotating mass is controlled by a separate governor by integrating the individual accelerating power. A general diagram of an individual power plant is shown in Fig.1

The simplified SFR model of the system is as shown in Fig.2, where all parameters are in per unit on an MVA base equal to the total rating of all generating units. The model behaviour depends on the following factors; the gain, $K_m$, the damping factor, $D$, the inertia constant $H$, the average reheat time constant, $T_R$, and the high pressure power fraction of the reheat turbines, $F_{H}$. The range of $T_R$ is about 6 to 12 seconds and tends to dominate the response of the largest fraction of turbine power output. Therefore we ignore all the time constants. The value of $H$ is on the order of 3 to 6 seconds for a typical large unit and is always multiplied by two, which increases its effect.

Notifications used in this figure are as follows:

- $H$: Inertia constant of a generator in per unit second
- $D$: Damping factor
- $K_m$: Mechanical power gain factor
- $F_{H}$: Fraction of the total power generated by the high power turbine
- $R$: Speed regulation or drop of the governor
Reheat time constants in seconds
Disturbance power in per unit
Accelerating power in per unit
Turbine mechanical power output in per unit

For this system model we compute the frequency response in per unit can be expressed as

\[ \Delta w = \left( R W_p^2 / DR + K_n \right) \left( 1 + T_s s \right) P_n / S^2 + 2 \zeta W_p s + W_p ^2 \]  

(1)

Where,

\[ W_p = DR + K_n / 2HR T_R \]

\[ \zeta = (2HR + (DR + K_n F_n) T_R / 2DR + K_n) / W_p \]

and the per unit speed or frequency can be computed for an disturbance power. For sudden disturbances, large or small, we are usually interested in disturbance power in the form of a step function \( P_{step} \) which change equation (1) into the following form

\[ \Delta w = \left( R W_p^2 / DR + K_n \right) \left( 1 + T_s s \right) P_{step} / S^2 + 2 \zeta W_p s + W_p ^2 \]  

(2)

\[ \Delta w(t) = R P_{step} / DR + K_n \left[ 1 + \alpha e^{-\zeta W_{同盟} [W_p t + \phi]} \right] \]  

(3)

Where,

\[ \alpha = \sqrt{1 - 2 T_s \zeta W_p + T_s \zeta W_p ^2} / (1 - \zeta ^2) \]

\[ W_p = W_p \sqrt{1 - \zeta ^2} \]

\[ \phi = \phi_1 - \phi_2 \]

\[ \phi_1 = \tan^{-1}(W_p T_s / [1 - \zeta W_p T_s]) \]

\[ \phi_2 = \tan^{-1}\left( \sqrt{\zeta - \zeta ^2} \right) \]

Here we are using SFR model to calculate \( P.L \Delta \), which is defined as the maximum amount of power mismatch (in p.u.) between the generation and load, for which the frequency does not fall below a present value. The study considers this value of the frequency as 58 Hz in 60 Hz system and 48 Hz in a 50 Hz system.

III. RESULT AND DISCUSSION

The frequency performance of the large power system can be represented approximately by a linear system frequency response (SFR) model. The basic SFR model averages the machine dynamic behaviour in a large system into an equivalent single machine. The SFR model examine only the frequencies associated with changes in the shaft speed. The time response is a damped sinusoidal, as shown in Fig. 3. It is usually recognized that sufficient load should be shed to limit the frequency decline to above 57 Hz. If we select a goal of 57 Hz as the maximum allowable operating frequency, we may estimate the maximum disturbance for a typical system from the plots of Fig. 3, which shows the frequency deviation without any load shedding. This indicates that, for load disturbance greater than about 0.4 per unit the frequency deviation is very likely to drop below 57 Hz. The amount of load shed and the timing of each step of load shedding must be determined. Some utilities set the first step of load shedding at 59.5 Hz. This is chosen for the reason that the large turbine-generator sets are not rated for continuous operation below about 59.5 Hz. The no. of steps at which load will be shed and the amount to be shed at each frequency step. The method of computing the settings of
under frequency load shedding by the following example. For that we have assumed the largest disturbance to be a sudden maximum disturbance at 100% of the total generation of the system. The curves of disturbances up to and including this 1.0 p.u. disturbance are shown in Fig. 3. We also observe that no load shedding is required for disturbances smaller than 0.4 per unit as these disturbances do not cause the frequency to fall below the 57 Hz threshold.

![Fig. 3 Frequency Response to Load Disturbance](image)

The average system parameters for this example are as follows:

\[
K_m = 0.85, \quad F_h = 0.3, \quad D = 1.0, \quad T_h = 8.0, \quad H = 3.5, \quad R = 0.06
\]

Table shows the computed initial slope and maximum frequency deviation for the typical system condition.

<table>
<thead>
<tr>
<th>P-Step (p.u.)</th>
<th>m_0 (p.u. / s)</th>
<th>Δω_max (Hz)</th>
<th>f_min (Hz)</th>
</tr>
</thead>
<tbody>
<tr>
<td>-0.2000</td>
<td>-0.0286</td>
<td>-1.6438</td>
<td>58.356</td>
</tr>
<tr>
<td>-0.3648</td>
<td>-0.0521</td>
<td>-3.0000</td>
<td>57.000</td>
</tr>
<tr>
<td>-0.4000</td>
<td>-0.0521</td>
<td>-3.2876</td>
<td>56.712</td>
</tr>
<tr>
<td>-0.6000</td>
<td>-0.0857</td>
<td>-5.1429</td>
<td>55.069</td>
</tr>
<tr>
<td>-0.8000</td>
<td>-0.1143</td>
<td>-6.5751</td>
<td>53.425</td>
</tr>
<tr>
<td>-1.0000</td>
<td>-0.1429</td>
<td>-8.2189</td>
<td>51.781</td>
</tr>
</tbody>
</table>

Table I Initial Slope and Maximum Deviation

The second row of Table represents the maximum step change of load that can be permitted if the frequency is not to decline below 57 Hz. Therefore, when the magnitude of the observed initial slope is greater than 0.0521 p.u. /s, load shedding must be triggered. Note that the initial slope for -0.3648 p.u. step disturbance is -0.0521 p.u. /s. So the load that must be shed by subtracting 0.0521 per unit/s from the computed slope for any disturbance. The size of the disturbance is the initial slope of frequency decline. This slope equal to

\[
m_0 = 60 \times \frac{Δ\omega}{dt} = \frac{60 \times P_{\text{Step}}}{2 \times H} \frac{\text{Hz}}{\sec}
\]

Where \(m_0\) is defined to be the initial slope in hertz per second. Suppose that this quantity is observed continuously, so that load shedding can be triggered when the slope exceeds some critical value. Knowing the slope, we can determine the size of the disturbance if the inertia constant is known. The value of inertia constant is approximately between 3 to 5 seconds. This permits us to quickly estimates the size of the disturbance to be,
The load shedding plan use a time delay of 0.1 sec. or 6 cycles, for each load shedding step. It is important to detect the need for shedding very quickly and to complete all load shedding before the frequency can decay to hazardous level. The load shedding is initiated at about 0.1 seconds after the disturbance, with the first step of load shed after 0.1 second time delay, or at about 0.2 seconds. Therefore, the incremental load shed should be equal to the following.

\[
P_{shed} = \frac{P_{step}}{2H} - 0.0521 \text{ p.u. sec} (6)
\]

Solving for \(P_{shed}\) and substitution in equation (5) for the step change, we compute,

\[
P_{shed} = P_{step} - 0.1042*H = H*(m_0/30 - 0.1042) \text{ p.u.} (7)
\]

From this equation, we may compute the incremental step function of load to be shed for various sizes of the initial disturbances. The amount of load shed in per unit is a function of only the inertia constant and the observed slope. The initial slope is the only unknown in equation (7). This initial slope contains all the information required to estimate the size of the step change in load caused by the system separation and to forecast the depth and even the approximate time of the maximum frequency dip. The amount of load to be shed is determined from equation (7). The result of load shed values and frequencies are as shown in table below.

<table>
<thead>
<tr>
<th>Frequency (Hz)</th>
<th>Amount of Load Shed (p.u.)</th>
</tr>
</thead>
<tbody>
<tr>
<td>59.5</td>
<td>0.317</td>
</tr>
<tr>
<td>59.2</td>
<td>0.10</td>
</tr>
<tr>
<td>58.9</td>
<td>0.10</td>
</tr>
<tr>
<td>58.6</td>
<td>0.10</td>
</tr>
<tr>
<td>58.3</td>
<td>0.05</td>
</tr>
<tr>
<td>Total load shed</td>
<td>0.667</td>
</tr>
</tbody>
</table>

Table II Load Shed Schedule for 1.0 per Unit Step

Using the load shedding steps shown in Table II and the given system parameters, the frequency response is computed using the SFR model. The result is shown in Fig. 4. It is very difficult to visualize the effect of each physical parameter of the SFR model without plotting the results. So, the each parameter is now varied and the results plotted to illustrate the effect of that parameters.

Fig. 4 Result of Load shedding according to Adaptive schedule
The effect of Governor Droop, R
The value of R is varied from 0.05 to 0.10 in the increment of 0.01 per unit. The results are shown in Fig.5. It is observed that the assumed droop setting R has no effect on the initial rate of frequency decline.

The effect of Inertia, H
The value of H has a direct effect on the initial slope, as shown in the Fig.6 for various H values. The inertia constant affects the values of $\omega_n$ and $\xi$ and also on the initial slope and the time of the peak response.

The effect of Reheat Time Constant, $T_R$
The reheat time constant is an important system parameter. Fig.7 shows a range that is typical of large generating units. $T_R$ has an effect on the damping ratio $\xi$ and the natural undamped frequency $\omega_n$ but does not affect the initial slope.
The effect of Frequency Dependence of Load

Power system loads are known to be sensitive to system frequency. The incremental change in load is a function of the incremental change in frequency. But this effect is included in the model in the form of the damping constant $D$, so we can write,

$$\Delta PL = D\Delta\omega$$  \hspace{1cm} (8)

Where all values are in per unit. The system undamped natural frequency and damping are function of $D$, but this dependence is appears as the product of DR. Since $R$ is small, nominally about 0.05 per unit, the product DR will be small and the effect of D is dismissed. The effect on the frequency response with varying $D$ is as shown in Fig.8. It is observed that the plot is similar to Fig.5 but with the reduced effect.

**IV. CONCLUSION**

The SFR model of system frequency behaviour following an event of large disturbance on the system is a simplified model of system behaviour. The model developed for this purpose omits many details and ignores small time constants to approximate the system frequency performance, including variation in different parameters. An adaptive scheme is
described for determining the amount of load to be shed by under frequency load shedding method based on the initial slope calculation. This initial slope contains all the information required to estimate the size of the step change in load caused by the system disturbance. From this observation, the first step of load shedding can be set up adaptively, with additional steps to follow at prescribed intervals and amount. The first step is considered the most important and is selected arbitrarily at one half the static load shed target, with additional increments of about 0.1 per unit to be shed at 0.3 Hz increments until the dynamic load shed amount has been reached.

REFERENCES


BIOGRAPHY

**Kinjal G. Shah** received the B.E. Degree in Electrical Engineering From A.D. Patel Institute Of Technology, V.V.Nagar, Gujarat, India in 2011 and M.Tech degree in Electrical Power System from Charutar University Of Science and Technology, CHARUSAT, Changa, Gujarat,India in 2013.She is currently an Assistant Professor in the Department of Electrical Engineering, Vadodara Institute Of Engineering, Vadodara, Gujarat. Her research interests include electrical machines, wide area monitoring systems, voltage stability assessment and control.

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